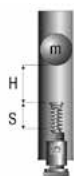


Hydraulic shock absorber selection

Five basic criteria are required for sizing the shock absorbers:

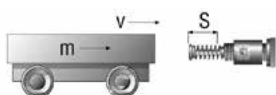
- impacting mass m (kg)
- impact speed v (m/s)
- additional external forces acting on the mass e.g. propelling force F (N)
- number of strokes of the shock absorber per hour X (1/h)
- number of parallel shock absorbers

Free falling mass



- $W_k = m \cdot g \cdot H$
- $W_A = m \cdot g \cdot S$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$
- $v = v_e = \sqrt{2 \cdot g \cdot H}$

Mass without propelling force



- $W_{kg} = \frac{m \cdot v^2}{2}$
- $W_{kg/h} = W_{kg} \cdot X$
- $v = v_e$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

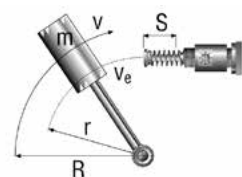
Mass with propelling force, horizontal



Movement downward: $W_A = (F + m \cdot g) \cdot S$
 Movement upward: $W_A = (F - m \cdot g) \cdot S$

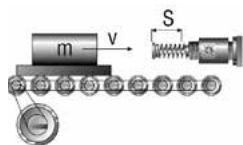
- $v_e = \frac{v}{K1}$
- $W_k = \frac{m \cdot v_e^2}{2}$
- $W_A = F \cdot S$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Swinging mass without propelling force



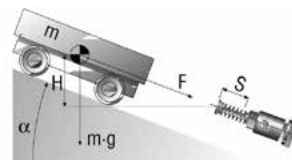
- $W_k = \frac{m \cdot v^2}{2} = \frac{J \cdot \omega^2}{2}$
- $W_A = \frac{M \cdot S}{r}$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $v_e = r \cdot \omega = \frac{v \cdot r}{R}$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Mass on driven rollers



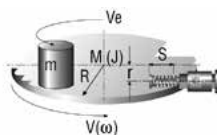
- $W_k = \frac{m \cdot v^2}{2}$
- $W_A = m \cdot g \cdot S \cdot \mu$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $v = v_e$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Mass on incline



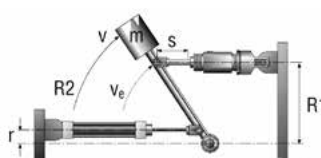
- $W_k = m \cdot g \cdot H$
- $W_A = m \cdot g \cdot \sin \alpha \cdot S$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $v = v_e = \sqrt{2 \cdot g \cdot H}$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Rotary table with propelling force



- $W_k = \frac{m \cdot v^2}{2} = \frac{J \cdot \omega^2}{2}$
- $W_A = \frac{M \cdot S}{r}$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $v_e = r \cdot \omega = \frac{v \cdot r}{R}$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Swinging mass with propelling force



- $W_k = \frac{m \cdot v^2}{2}$
- $W_A = \frac{M \cdot S}{R1} = \frac{F \cdot r \cdot S}{R1}$
- $W_{kg} = W_k + W_A$
- $W_{kg/h} = W_{kg} \cdot X$
- $v_e = R1 \cdot \omega = \frac{v \cdot R1}{R2}$
- $m_e = \frac{2 \cdot W_{kg}}{v_e^2}$

Formulae

Effective mass

$$m_e = \frac{2 \cdot W_{kg}}{v_e^2}$$

Counter force

$$F_G = \frac{W_{kg} \cdot 1.2^*}{S}$$

Deceleration time

$$t = \frac{2 \cdot S}{v_e} \cdot 1.2^*$$

Deceleration time

$$a = \frac{v^2}{2 \cdot S} \cdot 1.2^*$$

Stroke

$$S = \frac{v^2}{2 \cdot a} \cdot 1.2^*$$

*) Calculation for optimum setting. Allow a safety margin!

Used values and variables

W_k [Nm]	kinetic energy	K_1 [1]	correction factor for pneumatic drive force ($K_1=0.65$)
W_A [Nm]	propelling force energy	M [Nm]	torque
W_{kg} [Nm]	total energy	R, r [m]	radius
$W_{kg/h}$ [Nm·h ⁻¹]	total energy per hour	H [m]	height
m [kg]	mass	g [m·s ⁻²]	acceleration due to gravity
m_e [kg]	effective mass	J [kg·m ²]	moment of inertia
v [m·s ⁻¹]	impact speed	ω [s ⁻¹]	angular velocity
v_e [m·s ⁻¹]	effective speed	μ [1]	coefficient of friction (steel=0.2)
X [h ⁻¹]	number of strokes per hour	a [°]	angle
S [m]	stroke	a [m·s ⁻²]	acceleration / deceleration
F [N]	propelling force	t [s]	deceleration time
F_p [N]	pneumatic drive force	F_G [N]	counter force